Design of an affordable electric snowmobile — all over again

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ABSTRACT

The University of Alaska Fairbanks Nanook EV team's latest electric snowmobile has a 30 km (18.6 mi) range at 32 km/h (20 mi/h) under optimal snow conditions. Building on the 2009 competition success (winning Best Range with an Arctic Cat F7), we started this project with a much lighter chassis: a Ski-Doo Tundra 300F at 167 kg (370 lb), and a stronger motor, a NetGain WarP7 DC-series motor. The motor is connected directly to the sprocket shaft using two Gates Tripower BX belts. The accumulator is configured to support 177.6 V using 72 Turnigy 5 A-h lithiumion polymer hybrid cells, which utilize a gel electrolyte. This 7.992 kW-h battery pack meets the new rule requirement for the high voltage pack size. The batteries are connected to a 249 V Logisystems motor controller to power the motor. The snowmobile weighs 233 kg (514 lb) and has a top speed of 80 km/h (50 mi/h) and a noise level of 54 dB. This snowmobile is poised to do very well in the competition. Table 1 summarizes our goals.

INTRODUCTION

The National Science Foundation (NSF) supports research in polar regions, which are extremely sensitive areas that are highly impacted by pollution. In 2005 the Clean Snowmobile challenge added the additional category: "Zero-Emissions" in order to promote the use of vehicles which would not contaminate the fragile environments in these regions [1]. Also, it was important to avoid contaminating samples taken from these areas, as engine fumes could adversely affect the samples. Our team was also motivated to design an affordable electric snowmobile due to local high energy costs in

| Table 1: UAF Goals for CSC 2011 | | | | | | | | | |
|---------------------------------|------------|--------|-----------|--|--|--|--|--|--|
| Challenge UAF UAF | | | | | | | | | |
| Category | Record | Goal | Obtained | | | | | | |
| | | 30 km | | | | | | | |
| | 29 km | (18.6 | >30 km | | | | | | |
| Range | (18 mi) | mi) | (18.6 mi) | | | | | | |
| | | 227 kg | | | | | | | |
| | 226 kg* | (500 | <233 kg | | | | | | |
| Weight | (498 lb) | lbs) | (514 lbs) | | | | | | |
| 2.6 kN | | | | | | | | | |
| Drawbar | 2.56 kN | (590 | >2.6 kN | | | | | | |
| Pull | (575 lbf) | lbf) | (590 lbf) | | | | | | |
| Noise** | 65 dB | 64 dB | <64 dB | | | | | | |
| MSRP | \$14K | \$12K | <\$12K | | | | | | |
| *With 2.4 k | W-h pack | | | | | | | | |
| **With stuc | lded track | | | | | | | | |

Alaska. Gasoline is a precious commodity in rural villages across the state, many of which are not connected to a road system. The price of a gallon of gasoline can be in the \$10 range. Fuel is shipped to Alaskan villages in the summer by barge when the rivers and other shipping lanes are ice free. In some areas, fuel needs to be flown in, increasing the price even more [2].

The Nanook EV team has focused on finding transportation solutions for rural Alaska that can help reduce villagers' energy consumption, but still maintain their way of life. Electric vehicles have been a very promising solution when paired with locally generated renewable power. The team envisions clean, efficient electric vehicles used as primary local transportation, powered by renewable energy such as geothermal, wind and hydropower. These resources are abundant in rural Alaska but are currently under-utilized.

Our snowmobile is designed for the most practicality and performance that an electric sled can offer. At the same time, we strove to demonstrate that electric vehicles can be a viable option for certain applications. To accomplish this, a "better, faster, cheaper" design philosophy was adopted. The goal was to produce a system that had impressive performance, while still being affordable and easily accessible to the general public. This is our third year in this competition, and we offer an improved vehicle that is lighter and more comfortable for the rider, along with additional modifications to the original chassis, all while maintaining a clean, flexible, and aesthetically pleasing design.

Snowmobiles are an indispensable means of winter transportation in rural Alaska. While these machines are primarily used for recreation in the rest of the country, here they are an important tool that makes life in remote villages possible. Snowmobiles are therefore an ideal candidate for electric conversion. The Nanook EV team has extensive experience in converting traditional vehicles to run on electric power. Members of the team have converted everything from cars and trucks to ATVs and lawn mowers [3].

DESIGN STRATEGY

The main design strategy was to convert the snowmobile to be most successful at the competition. This year's competition scoring is more in line with National Science Foundation (NSF) contractor desires. Currently, over 57% of the events relate directly to their needs to support arctic research. The restriction of the accumulator size is an interesting complexity, however, the 8 kW-h size has been successful in doing 14 mi runs in Greenland. The acceleration event has been deleted and the objective handling event has been modified. Even though the acceleration event is gone, we still wanted a high power density battery for the straight-aways, which would benefit our machine on the objective handling track. We were unsuccessful in obtaining cutting-edge batteries that boasted mass energy density of over 300 W h/kg. These included sodium, semiconductor batteries. What we settled on was an inexpensive pack of lithium cobalt (LiCoO₂), which fit in nicely with our lightweight inexpensive chassis. This keeps our overall cost low for our final Manufacturer's Suggested Retail Price (MSRP).

Second, we had the incentive to keep the modification as simple as possible while using available and affordable parts. The parts needed to be low cost yet durable. Emphasis was added on using "off the shelf" components that are common to equipment such as electric forklifts and other electric vehicles. This not only would keep the MSRP low, but allow repeatability and ease for a pre-fabricated kit to be manufactured, so other users could enjoy and benefit from the use of an electric snowmobile. Although electric sleds have been emphasized in past competitions as tools for research purposes, our sled would also be ideal for the general public. Uses could include transportation to work in rural areas, checking trap-lines, subsistence hunting and fishing, and grooming ski and dog sled trails. We wanted a snowmobile that riders would want to use. Consumers are mostly interested in cost and range, and we feel we have achieved a snowmobile that meets those criteria.

Historically, it is interesting to note that dog mushing had been a common transportation choice in Alaska until the mid-1970s. Dog teams were used by the Postal Service in remote villages until the 1960s. The U.S. Army (Figure 1) and the National Park Service maintained sled dog teams for rescue missions or patrolling in remote areas. Denali National Park still employs a dog team today since motorized vehicles are not allowed in certain areas of the Park. With the advent of the snowmobile. Alaskans and other northern locals became enamored with this new machine; however, travel could be dangerous if the snowmobile broke down or ran out of fuel. An electric snowmobile offers a few different options. While the batteries may run out, the machine probably is not too far away from help. Also, the snowmobile could be equipped with an on-board emergency generator to provide enough power to limp home. But perhaps the most exciting option would be for the machine to



carry either a hydro or wind turbine that could be setup when the snowmobile was stationary. The use of portable solar panels was also considered, but solar is not a viable option for several reasons. Arctic regions enjoy little sunlight in the winter, and current solar panel technology is very inefficient and would require a trailer that is 12 m (40 ft) long to accommodate the number of panels required.

ENERGY STORAGE REVIEW

For the last two years to attain sufficient range, we placed great importance on obtaining the largest energy storage capacity possible. Batteries generally available for traction applications consist of metals such as lead, nickel and lithium. Thomas Edison designed the first traction batteries using nickel iron (NiFe) [4]. His battery (and his electric car) was later replaced with lead acid batteries (PbA) in the early 20th century. The nickel battery has evolved to such variants as the nickel cadmium (NiCd) and nickel metal hydride (NiMH) battery. Using nickel was an improvement over lead, except for cost and safety to the end user. Both lead and nickel exhibit a poor mass energy density of under 75 W h/kg. However, when using the lightest metal available, lithium batteries promised excellent mass energy density. At first, a non-rechargeable lithium battery was developed and dubbed "Lithium Metal". When the first lithium secondary cells were promoted, they were distinguished from non-rechargeable primary cells as "Lithium-Ion", or "Li-Ion." Today there are four major types of Lithium rechargeable batteries in production and available for resale. They are: lithium cobalt

(LiCoO₂), lithium manganese (LiMn₂O₄), lithium nickel (LiMn_xNi_yCo_zO₂), and lithium iron phosphate (LiFePO₄). Table 2 shows various cell attributes.

For the last two years our team has used lithium iron phosphate batteries. Using them kept our MSRP and our build budget low. At the time we decided they were a better alternative to lithium cobalt batteries because of its environmental benignity, its abundance, and the fact that it was less expensive than LiCoO₂. Additionally, the redox couple Fe⁺³/Fe⁺² is conveniently located at 3.45 V with respect to Li⁺¹/Li, and is compatible with many organic and polymer electrolytes. The successful commercialization of LiFePO₄ happened due to its high electrochemical performance, particularly in terms of reversible capacities. The initial problems of low electronic conductivity and low Li-ion diffusion rates have been improved significantly by co-synthesizing it with conductive sources, and making the particle size remains smaller, which resulted in diffusion path reduction [5].

This year we decided to use lithium batteries that are used by many Remote Control (RC) hobbyists. RC LiPo batteries are a hybrid lithium polymer battery. The correct name for this type of battery is lithium-ion polymer, but the battery world of today simply calls them lithium polymer even though they are not a true dry type LiPo battery.

By introducing a gelled electrolyte into the polymer, the ion exchange rate is improved. Since the electrolyte is gelled, there is less chance of leakage, but it is still flammable. LiPo

| | | Nic | kel | | | | |
|--|--------|--------|--------|--------------------|-----------------------------------|---------------------|--------|
| | Lead | | | | LiMn _x Ni _y | | LiPoly |
| Criteria | Acid | NiCd | NiMH | LiCoO ₂ | Co _z O ₂ | LiFePO ₄ | Hybrid |
| Mass Energy Density (W·h/kg) | 35 | 40 | 75 | 180 | 160 | 110 | 140 |
| Volume Energy Density (W·h/L) | 68 | 50 | 200 | 250 | 250 | 220 | 286 |
| Power Density (W/g) | 0.18 | 0.15 | 0.7 | 3 | 3 | 3 | 4.2 |
| Cycle efficiency (% charge/discharge) | 70 | 70 | 70 | 95 | 95 | 95 | 95 |
| Self-discharge (%/month) | 10 | 10 | 30 | 5 | 5 | 5 | 3 |
| Cycle life (total cycles) | 200 | 1000 | 500 | 500 | 500 | 2000 | 1000 |
| Current cost (US Dollar/W·h) | \$0.05 | \$0.23 | \$0.47 | \$0.60 | \$0.60 | \$0.31 | \$0.40 |
| Nominal Voltage | 2.1 | 1.2 | 1.2 | 3.7 | 3.7 | 3.2 | 3.7 |
| BMS Required | No | No | No | Yes | Yes | No | Yes |
| Environmental | Poor | Bad | Good | Average | Average | Good | Good |
| Cost based on cycle life x W-h of Lead | 1 | 0.7 | 1.3 | 1.75 | 1.75 | 0.2 | .45 |

Table 2: Battery Chemistry Examined

hybrids are not as dangerous as Li-Ion's but they can still catch fire or explode if over charged, shorted, or punctured.

When first introduced, LiPo batteries were more expensive than Li-Ion because they are more difficult to manufacture. Prices have dropped substantially. LiPo hybrids use the same flat cell structure as their dry counter parts meaning they have the same flexibility with sizes and shapes.

RC LiPo battery cell is packaged in a foil pouch coincidentally called a pouch cell. Figure 2 shows a 6 cell LiPo RC battery pack used in our Tundra. [6]

Our team chose the lithium-ion polymer for our electric snowmobile. Our primary reasons were mass energy density, availability and cost. These batteries are the least expensive lithium batteries available based on mass energy density. They are slightly heavier, and have less energy density and power output than lithium cobalt.

We installed 72 Turnigy batteries. These batteries have a 22.2 V nominal voltage, making the total pack size 177.6 V. Connecting eight of these batteries in series gives us the 177.6 V. We have a total of nine series strings which are then parallel connected. Each string has an individual fuse. [7]. Higher voltages allow a smaller amount of amperages, which produces less heat and less wasted energy.

The batteries are protected using a Chargery Power BM6 Battery Management System (BMS), as shown in Figure 3. These BMS boards output both high voltage and low voltage warnings. .6 volts. The use of the BMS will provide a durable battery system that is capable of 1 500 cycles



Figure 2: Turnigy 22.2 volt pack with packaging removed to show pouch cells. (Courtesy ypedal.com)



Figure 3: Chargery Power BM6 V3.0

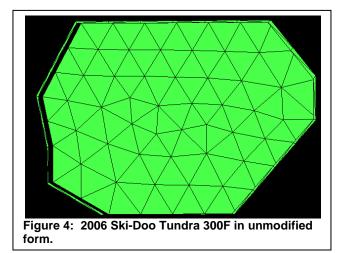
while only contributing about a 7% increase in the cost for the accumulator.

The batteries were confirmed to exhibit a low internal resistance during loading. Resistance values per battery are 0.003Ω . Having a low internal resistance allows the snowmobile motor to draw more power. This is a huge improvement when compared to lead acid batteries. Charger efficiency is also increased because less energy is wasted in heat.

ENERGY STORAGE CONTAINERS

In order to safely house all the batteries necessary to meet the range goal, a robust energy storage containment system was designed and fabricated. To ascertain whether or not the battery box could withstand any possible impact the snowmobile would experience, the battery box was computer modeled (figure 4) using 0.50" polycarbonate. This material has a young's modulus of 3.2×10^9 N/m, and a density of 1.20 g/cm. The box was modeled to withstand an impact at 20mph, with the 500 pounds of snowmobile compressing the box. This model shows that this material was strong enough to meet our needs. The results show that the box did not deform at all.

As with all the other components added, the location and construction of the energy storage containers contribute to the reproducibility and



practicality of the design.

DRIVE TRAIN

The gasoline engine is removed, along with the continuously variable transmission (CVT), the fuel tank, the muffler, and other associated parts. These are replaced with a Net Gain WarP7 DC series motor, a Logisystems Controllers 550 Amp 249 V Direct Current Controller, a 177.6 V battery system, and two Gates BX34 V-Belts.

Another way to increase the range of the design was to increase the drive train efficiency. The original CVT in the snowmobile was designed to keep the internal combustion engine operating at its optimal range. This however, is not an issue with an electric motor as a power source. Electric motors are capable of operating effectively at a much wider range of operating speeds. This property, combined with the ability of an electric motor to spin freely even when electrical power is not being supplied, allowed the use of a much more efficient direct drive system. Not removing the CVT can cause a decrease of performance by 20 %.

Removal of the chain case and jack shaft means that the snowmobile will require only two fluids: brake fluid and bearing grease. This makes a cleaner vehicle. It also reduces weight, and allows a simpler redesign.

With the chain case removed we had three choices for belts. These are Standard V-Belt, and two synchronous belts: Gates Polychain or a Goodyear Eagle Synchro belt (Table 3). The Synchronous belts afford a better efficiency of 98%, while the V-Belt slippage classified them with a 95 % rating. Synchronous belts also make 73 dB of noise whereas V-Belts are quiet. On the other hand, V-Belts cannot do as much power. In the end the design team went with the V-Belt for simplicity. We lost 3 percent efficiency by using the V-Belt; however, we can combat that by improving the motor efficiency. Thus, we installed

| Table | 3: | Belt | Desian | Criteria |
|-------|-----|------|--------|----------|
| | ••• | | - 00.g | ••••••• |

| Option | Cost | Simplicity | Eff. | Noise |
|--------------------|------|------------|------|-------|
| V-Belt | Low | Yes | Good | Quiet |
| Gates Polychain | High | No | Best | 73 dB |
| Goodyear Eagle | High | No | Best | 69 dB |

Helwig-Carbon red-top brushes, increased the spring pressure and used World Wide Bearings ceramic bearings. These design upgrades for the motor increased efficiency by 3 % and offset the losses of V-Belt usage.

The Red Top design is more efficient because it helps to make better contact with the commutator, producing less arcing and less Voltage drop. Less Voltage drop allows for more Voltage to get to the armature windings, making the motor more powerful and efficient. The Red Top design can be made with many different carbons and the composition of the brush material will also make a difference in the Voltage drop to the commutator and the motor efficiency. Brush grade H60 was chosen for cold weather applications. Because this traction motor is in a vehicle that is subjected to rough terrain spring pressure was increased from 6 to 9 Ib/in².[8]

Another factor that we considered in regard to belt usage is that synchronous belts require good alignment, otherwise the rubber erodes and leaves rubber debris on the frozen tundra, especially in arctic temperatures, when rubber tends to crack and break. This is not a good thing for a zero-emissions vehicle.

The design utilizes a standard industrial V-Belt system for increased efficiency over the stock CVT. By using a conventional V-Belt, it was possible to stay true to the low cost, easily reproducible design strategy. Using Gates Design Flex Pro software [9], the minimum pulley diameter and belt type was chosen. The BX type belt was selected due to its performance at higher rev/min service, which could be reached at the vehicle's top speed.

This belt can safely handle only half of the total motor output horsepower, so our application uses two belts.

In order to maximize range at speeds needed for the competition, sheaves which would allow the motor to operate at its most efficient speed were chosen. The ratio of the sheaves allow for 2 100 rev/min of the motor while traveling at 32 km/h (20 mi/h). This ratio puts the motor at its most efficient operating point, but still allows for higher speeds.

MOTOR

In making the motor selection we wanted the most reliable motor available. In keeping with our underlying design methodology, a DC motor will give more power per dollar then an AC motor. AC motor setups typically cost at least four times more than DC. NetGain Motors designs DC motors which are manufactured by Warfield Electric especially for the Electric Vehicle industry.

We chose a NetGain WarP 7" motor (Figure 5). It has a 181 mm (7.125 in) diameter and is 425 mm (16.75 in) wide. Thus, it fits nicely on the width of the tunnel. It weighs 45.5 kg (100.5 lb) and delivers a continuous power of 15.47 kW (21.75 hp). Additionally, the motor has the largest shaft diameter of 28.575 mm (1.125 in) in this size case. The lamination size is 16% larger than a 203.2 mm (8 in) diameter motor. Advanced timing is easily set with pre-drilled holes. We did advance time the motor to gain about 2 % efficiency.

The motor also exceeds "H" class insulation and can do 7 200 rev/min at 120 V and 400 A for five minutes. With our emphasis for the design to be the best at the 32 km/h (20 mi/h) range event, the motor will produce 47 N·m (35 lb-f) of torque, and 2 100 rev/min using 48 V and 230 A. We performed the multiplication of the voltage and the amperage to arrive at a power of 11 kW for this motor at that setting. This produces a mechanical power of 8.9 kW and is a sufficient amount of power for the range event based on the last two years of the Clean Snowmobile Challenge results.

CONTROLLER

We are using a Logisystems Controller 120/196 which is rated for 249 V and 550 A continuously (Figure 6). This is a serious controller capable of doing 137 kW continuously. It is a DC controller designed for brushed DC motors. It comes with 0-5 k Ω input from the throttle, and exhibits a 0.25 V drop on a 200 V input. Thus, it is 99 % efficient, except when the temperature exceeds 73°C, which will not occur in our design since we will only be using about 10 percent of its capacity. We also are using high volume delta fans to ensure proper cooling. This controller runs the motor with a Pulse Width Modulated at 14 kHz. It weighs 4.35 kg (9.5 lb) and is rated for -40°C (which occurs frequently in Alaska). It has come preset from the factory to allow effective



Figure 6: This is the DC brushed motor controller from Logisystems Controllers. It is manufactured in Odessa, TX and can do 137 kW continuous power. The three large connectors connect to the battery pack and motor. The three small connectors are for the variable resistor and the key switch.

acceleration.



Figure 5: Picture of Warp7 motor used in the Ski-Doo Tundra 300F. With the double shafted motor the brake rotor was installed on Commutator End (CE), left, and the pulley to the sprocket shaft was installed on the Drive End (DE), right.

COUPLER

We wished to remove as much rotating mass as possible in our electric snowmobile. The chain case was removed, and a 66 mm (2.6 in) Outside Diameter (OD) pulley was attached to the motor, and a 165 mm (6.5 in) OD pulley was attached to sprocket shaft (Figure 7). The gear ratio was found by determining the speed at which the track driver would have to turn in order to travel at 32 km/h (20 mi/h). Using the 203 mm (8 in) sprocket shaft, we calculated the circumference to be 638 mm (25 in) and then running the motor at 2 100 rev/min and then reducing that speed by 0.4 to 840 rev/min, the snowmobile will run at 32 km/h (20 mi/h). Using the motor curves in Figure 8 we found 2 100 rpm requires approximately 230 A at 48 V which is 11 kW.

MOTOR MOUNT

One obvious component deletion was the internal combustion engine. This deletion created a major design challenge: to develop an electric motor integration system. To maintain drive train efficiency, we designed a mounting system for the motor that integrates belt tensioning. The system design does not require the use of a tensioner or idler pulley, thus increasing both efficiency and belt life. It is very similar to an automotive alternator V-Belt tensioning

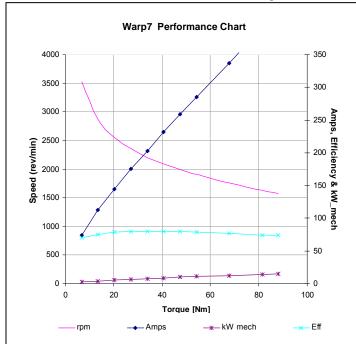


Figure 8: Torque-speed characteristics of the Warp7. Data was taken at 48 V and the load was varied from 6 N•m to 88 N•m. At each increment the motor speed (rev/min) and amperage were noted. Mechanical Power (kW) was calculated, and so was motor efficiency.

apparatus. The motor pivots about an axis to lengthen or shorten the distance between the two pulleys. The mount is constructed from 6061-T6 aluminum for long term durability -- as well as for its availability -- which is important to practicality and reproducibility. In order to produce a design

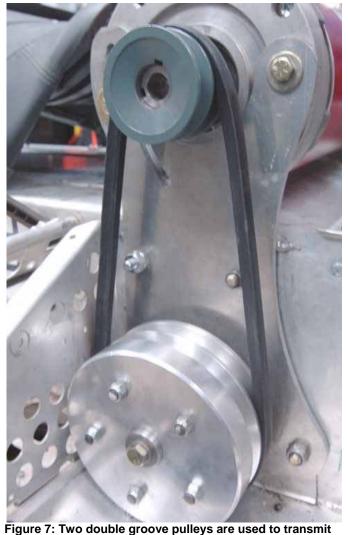


Figure 7: Two double groove pulleys are used to transmit the power from the motor to the sprocket shaft. Also pictured is the motor mount which is mounted on both sides of the tunnel.

for the mount, we had to determine the loads that the motor would produce. We accomplished this by examining the targeted performance for the snowmobile, and using those targets to identify the forces to be developed by the motor. After including the weight of the motor itself and doing an analysis for impact loading caused by bumps in the trail, we then included a safety factor, and were able to find the expected loads on the motor mount using static analysis. Once this was done, we were able to go about designing the mount.

Each component must be able to safely handle both the weight of the motor and the drive forces that the motor produces. In order to ensure that

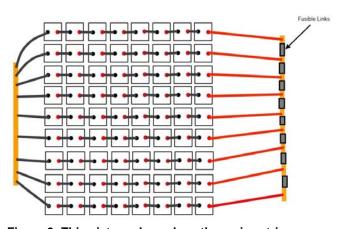


Figure 9: This picture shows how the series strings are paralleled at the bus bars and fused.

each component is up to the task, the Finite Element Analysis suite COSMOS (which includes the design program SolidWorks) was utilized. With this program, the stress distributions within the parts and with the expected loads could be calculated. Due to the accelerated timeline of the project, these results were only used to ensure that an adequate safety factor was present. Generous safety factors were allowed since the means to test the parts to failure (in order to confirm the COSMOS results) were unavailable. The material of choice for the components was aluminum; it is both lightweight and easy to work with, while still being strong enough to handle the load. Costs were kept low by keeping precision machining to a minimum. When mating the motor to the chassis, modifications of the snowmobile were kept to a minimum. Existing bolt holes were used when possible. The motor was mounted at the original location of the fuel tank and oil supply. The reasoning behind this was that the batteries and the motor have similar densities, and because of the cylindrical battery size, they would fit more compactly in the original engine compartment. The design allows for a very strong, easy to use, and low cost solution to integrating the electric motor into the snowmobile.

BATTERY SELECTION

The snowmobile uses an energy storage system consisting of 72 Turnigy lithium-ion polymer batteries. These batteries were designed for high "C-rate" RC applications. Each battery is 5 A·h, and can allow 100 A·h drain.

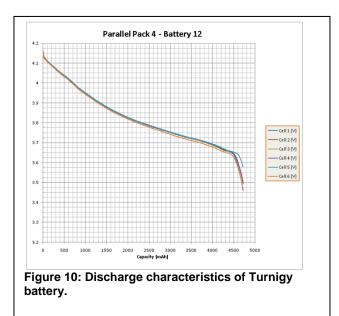
We selected these cells because their rapid discharge rates were five times as high as the Headway 38120 cells utilized last year. LiPoly batteries were also chosen due to their reasonable cost and weight. These batteries allow 1500 cycles when discharged at 80 % in each cycle. In Figure 10 you can see life cycle performance at various discharge cycles.

Table 4: Battery Pack Comparisons

| | Turnigy 22.2v 5000mAh | Change HeadWay 38120 |
|--------------------|-----------------------------|----------------------------|
| Battery Mass | 57 kg (125 lb) | 41% |
| Nominal Voltage | 177.6 V | 83% |
| Capacity | 60 A·h | 75% |
| Energy | 12.6 kW∙h | 63% |
| Energy Density | 100 W·h/kg | 140% |
| Power Density | 20 W/g | 400% |

After extensive battery research, we decided to use the cells manufactured by Turnigy. They were affordable and had a 40 % increase in mass energy density. There are currently many manufacturers of lithium batteries so our decision was extremely difficult and time intensive as we weighed our options.

Our pack consists of eight batteries in a series connection to produce 177.6 V and 5 A·h. Then nine of these strings are paralleled together making a total 177.6 V and 45 A·h. This pack is capable of producing a continuous 160 kW at 20 C-Rate.



In comparison to last year's machine [9], and conform this year's rules the energy storage was decreased by 63 % going from 12.672 kW·h to 7.992 kW·h. Using a 177.6 V will make the motor and controller run at lower amperages and be more efficient. The weight of the pack decreased by 240 % to 57 kg (125 lb) (Table 4).

BATTERY CHARGER

The team selected two UL-listed battery chargers that can charge half the pack at 100 V each. The Delta-Q Universal Input QuiQ is designed for a wide input voltage range from 85 to 265 VAC, allowing universal application. The 12 A maximum current draw ensures that the charger will work reliably, even through surges and sags. The QuiQ has a high efficiency design and the near unity Power Factor combine to make the QuiQ charger extremely grid-friendly. Over 88 % of power taken from the grid is converted to real power to charge the battery.

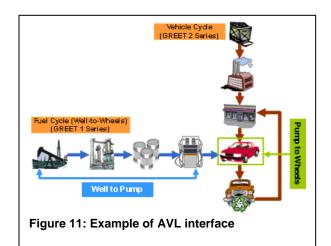
With the recent competition rule change requiring a UL-listed charger, we felt we surpassed that requirement as our selection is both UL and CE compliant and has passed stringent EMI, safety, vibration and water ingress protection (IP) tests. This charger offers leading edge efficiency, power factor correction and GFCI compatibility for safe and reliable operation.

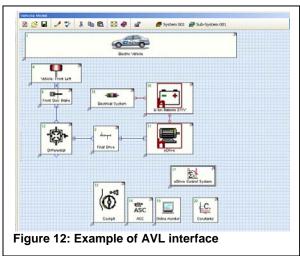
The QuiQ's intelligent microprocessor controller has optimized charge algorithms designed primarily for Lead Acid batteries. We selected an algorithm that would work with our Lithium-ion batteries. Utilizing the correct algorithms helps improve battery life and minimize maintenance. QuiQ is built for onboard operation in harsh environments. Its rugged, lightweight and intelligent design provides continuous operation in any application. High efficiency power conversion allows the QuiQ to be delivered in a fully sealed enclosure, making it ideal for onboard applications in the dirtiest and wettest environments. Reliability is increased by the reduction of moving parts.

ENERGY EFFICIENCY

To evaluate the efficiency of the Nanook EV, a comparison analysis with a standard production snowmobile was used. Assuming the best mileage a production IC snowmobile gets is 8.5 km/L (20 mi/gal), driving 32 km (20 mi) uses about 125 000 Btu of fossil fuel. This translates to 125 000 Btu / 32 km, which is 3 906 Btu of fossil fuels per km. The electric snowmobile averaged 500 W·h/km (800 W·h/mi) total energy use, which includes charging the batteries. Converting to British thermal units by multiplying 0.5 kW·h/km with 3 412 Btu/kW·h to obtain 1 706 Btu/km. Looking at how the electricity is generated will give a more accurate Btu comparison value, unless the sled can be recharged using alternative energy such as wind or solar power.

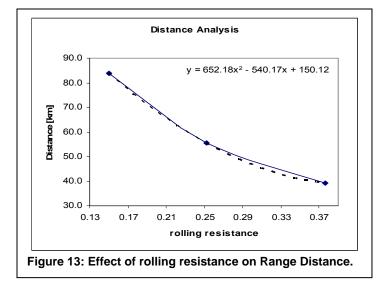
The worst-case scenario would be electricity from a coal fired power plant with an efficiency of 33 %. The fossil fuel input is 3 times the electrical power output, i.e. 3×1706 Btu/km = 5 118 Btu/km. This number shows that an electric snowmobile is less efficient than a production gasoline sled. However, if a more efficient power generation is used such as a 45 % efficient power plant, then 2.22 x 1706 Btu/km = 3 971 Btu/km which is similar to the original gasoline consumption. The fact that a typical electric vehicle still has a significantly shorter range demonstrates the large discrepancy in energy density from a gasoline-powered sled to an electric sled.





Also, it is interesting to point out that even if the energy consumption is the same in either using gasoline or electricity to power a snowmobile, there are additional energy needs in order to bring that energy to a gas tank or a wall outlet. Argonne National Laboratory's The Greenhouse Gases, Regulated Emissions, and Energy Use in Transportation (GREET) Model [11] can do a Fuel Cycle analysis, also known as "Well to Wheel" (or with snowmobiles "Well to Track.") (Figure 11).

This modeling software allows researchers and analysts to evaluate various vehicle and fuel combinations on a full fuel-cycle/vehicle-cycle basis. We used this modeling software to compare snowmobile combustion vs. electric snowmobiles. We estimated that an electric snowmobile operated with an 11 % reduction in CO_2 emissions and a 10 % reduction in Greenhouse Gases (GHG) based on energy generation in Fairbanks, AK [12]. The software will also give you modeling data on other emissions as well.



AVL has a software package called CRUISE for modeling the vehicle's powertrain efficiency. AVL CRUISE supports everyday tasks in a vehicle's system and driveline analysis throughout all development phases, from concept planning through to launch and beyond. Its application envelope covers conventional vehicle powertrains through to highly-advanced hybrid systems. It performs in all fuel economies and performance tests in a single run with the same vehicle model. We did some initial computer modeling with this software. In Figure 12 we modified an electric car example to be an electric snowmobile. We hope to share more results during the Oral Presentations.

| | Rolling | Distance | | | | |
|-----------|------------|----------|------|--|--|--|
| Snow | Resistance | _ | _ | | | |
| Condition | [rr] | km | mi | | | |
| Slush | 0.377 | 39.2 | 23.5 | | | |
| Ice | 0.252 | 55.4 | 33.3 | | | |
| Powder | 0.15 | 83.8 | 50.3 | | | |
| | 0.10 | 00.0 | 0010 | | | |

Table 5: Rolling Resistance Effect on Range

RANGE

For the past three challenges we have focused more on the range event than anything else. With the recent energy storage restriction more emphasis it has become harder to justify this focus. So for a vehicle to be practical it must be able to transport people and cargo over a usable range. There were many design decisions made to reach this goal. We didn't achieve our goals in 2009 since we expected to travel another 50 % further. What we didn't anticipate were extreme wet snow conditions. We have classified snow into three categories as shown in Table 5: Slush, Ice and Powder [12, 13]. Using data from the last two years of the CSC and Auth's Thesis [14] we calculated a rolling resistance coefficient. We also show our range estimation for our current sled depending on conditions.

Also, in Figure 13 we plotted Distance vs. rolling resistance which shows how the rolling resistance coefficient affects the overall distance performance of an electric snowmobile at 32 km/h (20 mi/h). Additionally, we looked at the force analysis to propel a vehicle. Typically there are four major criteria: Acceleration, Rolling Resistance, Hill Climbing and Wind Drag. Since the range event is on a level track we did not calculate Hill Climbing force. The other forces were calculated and the summation is shown in Table 6 as Required [W·h] in Column 5. We had

| Table 6: Using Calculated and Actual Ran | ge to Determine Rolling Resistance Coefficient |
|--|--|
|--|--|

| Team | Rider [kg] | Sled [kg] | Total [kg] | Required [W-h] | Battery Capacity [W-h] | Drive losses (-20%) | Range Calculated [km] | Range Actual [km] | Error | [W-h/km] | rr |
|--------|---------------|--------------|---------------|-------------------|------------------------------|---------------------------|-----------------------------|-------------------------|-------|----------|-------|
| UAF | | | | | | | | | 0.94 | | |
| '09 | 50 | 327 | 377 | 8747 | 8991 | 7193 | 26.5 | 26.7 | % | 872 | 0.377 |
| UWM | | | | | | | | | 1.17 | | |
| '09 | 68 | 322 | 390 | 8811 | 6750 | 5400 | 19.7 | 20 | % | 875 | 0.377 |
| McGill | | | | | | | | | 7.51 | | |
| '09 | 68 | 226 | 294 | 6422 | 2604 | 2083 | 10.4 | 9.7 | % | 698 | 0.377 |
| UWM | | | | | | | | | 0.13 | | |
| '08 | 68 | 313 | 381 | 5422 | 6092 | 4873 | 28.9 | 29 | % | 544 | 0.252 |

to add more acceleration events in the 2009 data because a track was not used. When we calculated [17] the rolling resistance force, we left the coefficient as a variable and calculated it last using the actual range data from four snowmobiles. We used four sleds: our Nanook, a UWM, a McGill from 2009, and a UWM from 2008. This gave us a good amount of data to compare.

There are some assumptions that we had to make: we estimated the rider's mass, and assumed 20% driveline losses for all teams. This would include both mechanical and electrical inefficiencies. We also had to look at the battery data given in previous design papers to obtain an idea of what size battery packs each school utilized. The other unknown was how much of the energy storage system each school optimized. In the end, we felt that the error between theoretical and actual range was small enough to obtain a relevant rolling resistance coefficient. We believe this data analysis is vital for future snowmobile modeling and design. Our team struggled in 2009 without this information. We felt confident that if we had the largest battery pack we would win best range. Both UAF and UWM had similar energy consumption of 872 W·h/km and 875 W·h/km respectively, and yet the team with the larger battery pack had the best range.

RANGE TEST

The snowmobile was driven on a 1.33 kilometer (0.83 mile) track for range testing. The sled was driven at a constant speed of 32 km/h (20 mi/h). We kept the speed constant to maintain zero acceleration as much as possible. We ran the machine until the BMS alerted us that the machine needed to be recharged. We obtained 30 km (18 mi) on hard-pack snow, which we calculated to be 266 W·h/km by dividing 7.992 kW·h and the 30 km. This exceeds the old 16

kilometer (10 mile) standard which is still listed as a design criterion in the Clean Snowmobile Challenge rules, and this range can be exceeded or reduced with different snow conditions. [1] Knowing this, we tested again on a warmer day and ran the sled out on a river's surface; we were able to drive 20 km (12.42 mi) before receiving the BMS warning. This was calculated to be 400 kW·h/km. This amount is much higher than the power draw we saw in the first test, and shows the large amount of variability that exists due to snow conditions.

DRAWBAR PULL TEST

The drawbar pull is an interesting event in that many of the qualities that lead to drawbar pull success can be detrimental to performance in other events. Chief among these qualities is weight. A heavy snowmobile will achieve lot of traction, and thus be able to pull more. On the other hand, that weight is cumbersome in events like the range and acceleration tests.

Judging from real-world experience, it was apparent that the limiting factor in the event would not be power, but traction. We may use studs to increase performance depending course conditions. To test the snowmobile's performance in the drawbar pull, the back end of the snowmobile was attached to the back end of a truck with a tow strap and force meter. The truck was shifted into neutral, and the snowmobile began pulling it at 6.4 km/h (4 mi/h).

Once this speed was attained, the brakes on the truck were slowly applied to progressively increase the resistance the snowmobile was pulling against, until traction was lost. The highest measured force was recorded. The force meter used was simply a piece of 25.4 mm (1 in)

by 6.35 mm (0.25 in) aluminum with a strain gauge attached.

During testing, the maximum recorded force pulled against was 2.6 kN (590 lbf) At this point, the track lost traction and began to spin out. The consistency of the snow at the test site was a loosely packed, dry powder. Loss of power was not a limiting factor during the test. Maximum pulling force can easily be improved with a different snow consistency.

NOISE

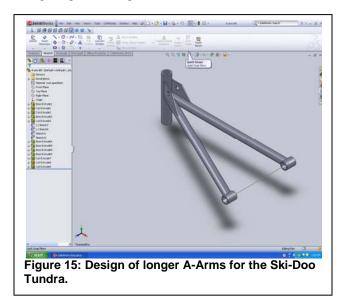
The overall sound output of the machine was found to be quite minimal. We experimented with different tracks from Camoplast and Kimpex and on light powder the sled was performing below 65 dB. To address subjective sound quality, the motor used this year has an internal fan which is much quieter. We also added a paddle damper to reduce the noise levels from the track and sprocket shaft. Also, as mentioned earlier we went with the quiet V-Belt for our traction system.

SUBJECTIVE HANDLING

With the goal of designing a sled for general recreational use, much importance was placed on the snowmobile's handling. The Tundra has a narrow ski stance of 812 mm (32 in) which makes it prone to being tippy. In order to receive a higher score at this year's competition, we would like to make the machine more stable. To accomplish this, all of the batteries are moved into the engine compartment along with another modification. It was decided that the A-arms of the snowmobile would be lengthened to create a wider ski-stance. The 2006 Skandic Tundra had a narrower ski stance, at 32".

Machines with narrower ski stance are used more for powder or deep snow riding, which makes it easier to carve (or lean) the snowmobile into turns. Trail machines are designed with a wider ski stance for comfort and stability. The A-arms should also make the snowmobile more stable, which would be accomplished by making them wider. As a result, we will also need to replace the current shocks with shocks that have a longer eye-to eye length. The shocks on the machine currently have an extended center-to-center length of 14.5." We will choose a pair of shocks with an extended center-tocenter lengh of 18" or greater. In addition to the shocks, we also need longer steering rods to reach the wider A-arms. According to the SAE

Clean Snowmobile Challange rules, the total travel of our suspension must be 6" or greater. It is necessary to ensure that we have shocks that are not too short, which would cause the machine to bottom out easily and violate the rules. On the other hand, it is also necessary to ensure that the shocks are not too long so the ski stance becomes narrow again and nullifies our efforts. We kept the center of mass as low as possible to keep the sled from having any tipping issues. Figure 15 shows the use of Solidworks in designing the longer A-arms.



The additional weight added to the snowmobile resides in the engine compartment. This allows for a low center of mass that the team wanted. The sled responds instantly to throttle input, a benefit associated with electric motors. Increases in speed can be made smoothly and quickly without the hesitation or 'jerking' often attributed to CVT clutches found in a traditional snowmachines. The sled is geared primarily for range by running the motor at its optimum rev/min while turning the track at a speed of 32 km/h (20 mi/h). As a result it can't pull the skis off the ground during rapid accelerations, but it does have enough torque to start and maneuver through relatively deep powder.

BRAKING- The machine still employs the stock hydraulic disk brake system mounted on the Commutator End (CE) shaft. Since these brakes were engineered to slow the original 295 kg (650 lb) sled from speeds in excess of 80 km/h (50 mi/h), they exhibit excellent performance while slowing the new additional weight. In preliminary acceleration tests, where quick emergency style braking was required, the brakes showed little or no sign of fade. The stock rotor never showed signs of excessive heat build up.

BALANCE – The snowmobile is well-balanced front to back and side to side. Since the gas engine and clutches were spatially replaced with a motor and battery pack that weighs more, the weight over the front skis is greater than the stock values. This allows for better steering of the snowmobile. Last year we learned that having too much weight over the track was not helpful in the subjective handling test.

OVERALL HANDLING -

The snowmobile exhibits a high overall level of comfort and performance. The seat is slightly elevated to simulate the popular high-rise aftermarket seats, decreasing the angle of the rider's knee and thus reducing joint and leg fatigue. The gauges are located in the stock locations, which still permits easy visual access. The original cable style throttle block was removed in favor of a resistor trigger, which is more comfortable, reducing wrist and thumb fatigue, which is common on traditional snowmobiles. While the power was reduced and the weight was increased, the sled is still enjoyable to ride. It is by no means bulky or sluggish as many would envision an electric snowmobile to be. Aesthetically, it still retains its performance oriented styling and stance. Although some snowmobiles are used in commercial or research applications, the majority of the market is driven by recreational consumers. With this in mind we feel it is important that our final result still retained its



Figure 16: 2006 Ski-Doo Tundra 300F in unmodified form.

original ability to provide a fun and comfortable ride, which the Nanook EV Two surely does.

The published dry weight for the original sled is 167 kg (370 lb). Filling all the fluids adds conservatively 36 kg (80 lb) bringing the total to 203 kg (448 lb). The Nanook EV tips the scales at 233 kg (514 lb). The net weight increase is a mere 30 kg (66 lb). This new weight allows our machine to be competitive with many four-stroke gasoline powered snowmobiles available. The team did some weight calculations to determine how weight affects range. It appears about 27 kg (50 lb) can reduce range by 3 km. We weren't happy with the battery pack energy storage capacity, but it may prove to be an adequate compromise. In the future the team will use more exotic materials to lighten the sled, and attempt to find a lighter battery pack with even higher energy density.

ACCELERATION

The acceleration rate is very challenging for an electric snowmobile. Although the acceleration event was deleted from this year's competition, acceleration performance is what the public would like to know. This is unfortunate because running the snowmobile at a faster speed hurts the possible range. The high power demands of the event require high electrical currents being fed to the motor (upwards of 600 amps), and the large forces involved push the mechanical components to their limit. As with the drawbar pull event, traction is a major concern, though not as critical.

The most important aspect of optimization for this performance is adequate motor sizing and gear selection. If the motor is too small, then the snowmobile will not be able to meet the minimum performance criteria for enthusiasts. If the motor is too large, the snowmobile may do well in the acceleration event, but the excessive loads that it places on the electrical system will hurt its performance in the distance event and harm its long-term durability. We believe the motor we selected, at 15.47 kW (21.75 hp), is the perfect size to provide both versatility and performance.

OBJECTIVE HANDLING This year's modified event requires towing a 454 kg (1 000 lb) sled through a course for time. We tested the sled with this weight, and found no issues so far. We let our riders do several practice runs since this event will rely greatly on driver skill and experience.

WEIGHT

COST

One advantage in working on a limited budget during this project is that our Manufacturer's Suggested Retail Price (MSRP) is extremely low. We went with a brushed DC motor to save \$3,000 off the final price. We used lithium polymer batteries to save another \$2,000. We used stock V-Belt Pulleys to save another \$700. This \$5,700 in savings should make us competitive against other teams, and make more researchers interested in acquiring a machine.

The Challenge rules have been adjusted this year to reflect representation of costs. However, recent commercial snowmobile pricing has been on the rise for the last several years. This makes most chassis used in 2010 prohibitively expensive to convert to electric. We are thankful that the rules allow for a credit on the original motor; however this is not a realistic if you were planning a conversion business. Unfortunately none of the four major snowmobile manufacturers have taken an interest in a commercial electric sled. We realize there are major shortcomings in electric snowmobiles for certain user groups. However, a recent start-up company named Premier Recreational Products and other overseas vendors have developed a gasoline powered family-sled for under \$4K [18]. Even though it is a smaller a "3/4" sled, the 96" track would be usable in many situations. Using a chassis like this in a conversion would have an instant weight savings, and would be less expensive overall to convert.

PREVIOUS CHALLENGE RESULTS

The track conditions in 2010 were very poor, and we did some quick analysis that showed that we had better data in 2009. During the 2009 Clean Snowmobile Challenge, the Nanook EV performed admirably. All components performed as designed, and in some instances performed better than expected.

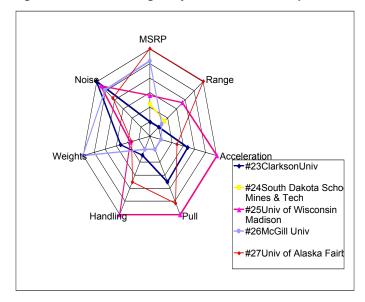
The first test in the competition was the range event. As range was a major focus for the design of the original Nanook EV, this test was critical to validate many of the design choices. During the test, the snowmobile was able to cover 27.6 km (16.6 mi). This distance was shorter than the anticipated range; however, it was still within the range simulations. Given the fact that the conditions during the test were not ideal and that all the other teams saw reduced performance compared to previous years, it is reasonable to assume that the reduced range was a result of the conditions, not problems with the vehicles' systems. The Nanook EV's performance in the range test was the best in the competition. The next farthest range was achieved by the UWM team with 20 kilometers (12.4 miles).

During the draw-bar pull test, the Nanook EV was capable of pulling against a force of 2.437 kN (548 lbf). This was much larger than the predicted result. We believe that our load cell used for measuring during initial testing may have been faulty. Our performance earned second place, behind UWM's result of 2.557 kN (575 lbf). This result is excellent considering the fact that the Nanook EV was not equipped with studs, which would have greatly increased traction in the competition conditions. In the objective handling event, the Nanook EV was able to place second despite lesser acceleration performance than some of the other sleds. This was largely due a combination of the well tuned suspension, easy handling characteristics, and high performance skis.

In addition to these competition highlights, the Nanook EV was also the least expensive snowmobile present. This was great validation of the design, considering that cost effectiveness and maximum range were the two primary design goals, and also the two events we won. The Nanook EV finished second overall in the competition, a great performance for a rookie team. A graphical summary of the competition performance compared to the other teams is shown in Figure 17.

SUMMARY

Having completed testing and competition with the Nanook EV 1 and 2, it is clear that the design goals were met. We had a successful 2009 season, an interesting rebuilding year in 2010, and with our current testing, we feel confident of success in March 2011. A zero emissions snowmobile that is capable of excelling in the areas of range, pulling power, noise, handling, and weight has been produced once again, and this machine can have a broad range of uses outside the scientific research market. The Nanook EV2 is a low-cost, durable, easily reproducible snowmobile that is a pleasure to ride. We believe we have developed a breakthrough product that will address some of the criticisms of electric snowmobiles. Figure 17: 2009 Challenge Objective results radar plot:



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ACKNOWLEDGMENTS

University Alaska Fairbanks - College of Engineering and Mining & Institute of Northern Engineering, University Alaska Fairbanks Alumni Association, University Alaska Fairbanks Provost's Office, U.S. Fish and Wildlife Service, Logisystems Controllers, NetGain Motors, ASME Northern Alaska Subsection, Kimpex, Lynden Freight, HMK, Helwig Carbon, World Wide Bearings, National Science Foundation, Northern Power Sports, Solidworks, AVL, Gates, Spangs Fab, Fairbanks City Bed-Tax, Usibelli Foundation, AWIM, John Deere, Curtis