Michigan Technological University

2013 SAE Clean Snowmobile Challenge Design Presentation

Presented by:

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Presentation Overview

Snowmobile Operator Design Intent

- ✓ Rider-Forward Ergonomics
- ✓ Quiet, Fuel Efficient 4-Stroke

Snowmobile Dealer Perspective

- ✓ Low Maintenance 4-Stroke Option in New Chassis
- ✓ Polaris Product Line Parts Availability
- ✓ Hayes Anti-lock Braking System

Environmental Protection

- ✓ Catalyst Selection for Emissions Reduction
- ✓ Sound Reduction with Exhaust Simulation
- ✓ Performance Electronics ECU
- ✓ Flex-Fuel Implementation
- ✓ Fuel Economy Estimates

Engineering Tests / Proof of Design

✓ Over-structure and Engine Mounts (Finite Element Analysis)

Conclusion

Questions/ Additional Information

Snowmobile Operator Design Intent

2013 Polaris Indy SP

- ✓ Rider-Forward Ergonomics
- ✓ Latest Market-Available Chassis
- ✓ Low Initial MSRP



2013 4-Stroke Polaris 875cc Pro-Star

- ✓ RZR Throttle Bodies
- ✓ MTU Designed Intake System
- ✓ Fuel-Efficient Ranger Camshafts and Combustion Characteristics
- ✓ Dependable Wet-Sump Oil System
- ✓ Tolerant to Ethanol Blends
- ✓ 55 HP Power Output



Snowmobile Dealer Perspective

- ✓ Low Maintenance 4-Stroke
 Option in Pro-Ride Chassis
- Viable Improvements in Power Output
- ✓ Alternative Fuel Option
- ✓ Hayes Anti-Lock Brake System for Controlled Braking in Extreme Conditions
- ✓ Reasonable MSRP of \$12,038.17



Environmental Protection

Catalyst Selection for Emissions Reduction

Catalyst Selection

- ✓ Three Catalysts Compared
- ✓ V-Converter 3-Way Chosen for: 79% Reduction in HC 23% Reduction in CO Slight increase in NOx

Catalyst Characteristics

2 Ch

- ✓ 63 C.I. Volume Sized for Pro-Star Engine Application
- ✓ Operates Most Efficient Near Stoichiometric AFR

Hydrocarbon Emissions

Environmental Protection Sound Reduction – Muffler Design Considerations

- Dual Chamber Muffler Analysis
- MATLAB Simulation to select optimum volume for transmission loss
- Packaging Within OEM Plastics

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Environmental Protection Sound Reduction – Final Exhaust Design and Validation

MTU Designed and Fabricated Exhaust System

- 1. 2 to 1 Collection Chamber
- 2. V-Converter Catalyst
- 3. Two-Chamber Muffler

Sound Reduction Verification

- ✓ 74 dB per J192 Standard
- ✓ Intake Noise is Main Focal Point for Improvement

Environmental Protection Engine and Fuel Management

Performance Electronics PE3

- ✓ Standalone Engine Control Unit (ECU)
- Precise Fuel and Ignition Control Based on 1250 Load Breakpoints
- ✓ Closed-Loop Fuel Control
 - Stoichiometric AFR Targeted
 - Long Term Factor Feedback
- ✓ 30+ User Defined Fuel and Ignition Trim Tables
- ✓ On-Board Wireless Tuning Capabilities
- ✓ Power output of 55 HP

Environmental Protection Flex Fuel Implementation

E40 – E85 Ethanol Compensation

- ✓ Siemens Ethanol Content Analyzer
- ✓ Zietronix A/D Converter
- ✓ P.E. Digital 0-5v input
- ✓ MTU Generated Fuel and Ignition Trims
 - Modeled with 33% fuel conversion efficiency at full load
- ✓ Ignition Trims Based on Ethanol Octane Content
- ✓ Control Algorithm for Varying Stoichiometric Air/Fuel Ratios Based on Ethanol and O₂ Sensor Input

Environmental Protection Calculated Fuel Economy

Fuel Economy Calculations for 45 mph Derived From:

- ✓ Injector flow rate
- ✓ Varying heating values and densities for E40 E85
- ✓ Observed Engine Speed and Injector Duty Cycle at Cruising Speed

Estimated Fuel Economy: 14.9 – 20.3 mpg

Engineering Tests / Proof of Design Engine Mounting Design and Verification

- Engine Securely Cradled at Three mounting Location
- ✓ Torque Stop on PTO Side of Engine

Engineering Tests / Proof of Design Engine Mounting Design and Verification

- ✓ Maximum Von Mises Stress = 23.29 ksi
- ✓ Yield Strength of Aluminum 6061-T6 = 40 ksi
- ✓ Safety Factor = 1.72
- ✓ 300+ Miles of Field Verification

Engineering Tests / Proof of Design Over-Structure Modification and Verification

Over-Structure Required Modification for Engine Fitment

- ✓ Maximum Von Mises Stress: 6.275 ksi
- ✓ Safety Factor = 6.37
- ✓ Maximum Displacement = 0.019 Inches

Von Mises Contour Plot

Displacement Contour Plot

Conclusion

- ✓ Unique Engine and Chassis Configuration
- ✓ Desirable Characteristics for both Consumer and Dealer
- ✓ Environmentally friendly
 - Reduced Emissions Through Catalyst and Engine Calibration
 - Reduced Exhaust Noise Emissions with Dual-Chamber Exhaust System
 - Flex-Fuel Implementation
- ✓ All Chassis Modifications Necessary for Engine Implementation Verified Through FEA and Field Testing

Questions?

Additional Information

Calculating Loads

 $F_{friction} = Belt_{tension} = \frac{Torque}{r_i} = \frac{65}{0.048} = 1354 \ lbs$ Impact Loading Factor = $1 + \sqrt{1 + \frac{v^2}{g} * \delta_{st}}$

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Additional Information

FEA

Loads

Belt tension calculated using torque of motor Static weight of motor = 110 lbs Impact Load due to a "suddenly applied load" Boundary Conditions Fixed at chassis connection for front mount and

where rear mounts connect to the isolators

Final Engine Mounting System

Rear Engine Mounts

Front Engine Mount

Torque Stop

Additional Information

Fuel Economy and Ethanol Trim Verification

1 Injector Flowrate (cr/mi) 100 for 3 bar of pressure (cm/mi) QHV, Gas 4730 kl/g (cm/mi) <	R .
2 Injector Flowrate (m3/s) 1.67E-06 Impactor Flowrate (m3/s) Impactor Flowrate (m3/	
3 Density Gasoline (lb/ft ³³) 44.9 44.9 64.9 64.0	
4 Density Gas (kg/m^3) 719.298 Cond Cond Cond Cond Cond For 45 mph Cond Cond 5 Density Ethanol (kg/m3) 789 Cond Cond <th< td=""><td></td></th<>	
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18	
19	
20 E60 Open Time (Mass Flow Rate (kg/s) kg fuel injected open time/s kg/s Speed 45 mph	
21 0.0025 0.001247789 3.11947E-06 400 0.00125 0.0125 mile/s	
22 0.00193 kg/s	
23 Fuel efficiency, E60 0.5709535 0.00379 kg/m^3 to kg/gal	

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Additional Information

Performance Electronics ECU

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Emissions

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Emissions

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Collection Box Failure

• Collection box failure during validation testing

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- Re-designed collection box after failure analysis
 - Features chamfered edges for improved flow characteristics and heat management

Muffler Design

- CAD model of outer shell
 - Designed to be functional yet easy to fabricate/manufacture

- Matlab analysis of single chamber transmission loss
 - Each red spike represents a predicted resonance if aligned with a blue engine frequency line

- Matlab analysis of dual chamber transmission loss
 - Analyzed at 900°C, 950 °C, and 1000 °C
 - Muffler simulations were ran for chamber distance of 5, 6, 7, 8, and 9 inches from the top of the muffler

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